



ADDENDA

**ASHRAE Addendum e to
ASHRAE Guideline 36-2018**

High-Performance Sequences of Operation for HVAC Systems

Approved by ASHRAE on August 16, 2019.

This addendum was approved by a Standing Guideline Project Committee (SGPC) for which the Standards Committee has established a documented program for regular publication of addenda or revisions, including procedures for timely, documented, consensus action on requests for change to any part of the guideline. Instructions for how to submit a change can be found on the ASHRAE® website (<https://www.ashrae.org/continuous-maintenance>).

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- b. participation in the next review of the Guideline,
- c. offering constructive criticism for improving the Guideline, or
- d. permission to reprint portions of the Guideline.

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FOREWORD

In the current edition of Guideline 36, designers must determine V_{min} , the VAV box minimum for all terminal types. This was left as a variable, rather than automatically calculated by the sequences, primarily because the minimum for systems complying with ASHRAE Standard 62.1 is not readily calculated when using Standard 62.1, Section 6.2.5, and the Multiple Spaces Equation (MSE) in Appendix A. It must be determined by iterating on airflow rates and occupancy assumptions in each zone of the system, typically using the 62MZCalc spreadsheet provided with the Standard 62.1 User's Manual.

Standard 62.1-2016, Addendum f, approved and published in May 2018, created a Simplified Procedure for determining outdoor air rates for multiple-zone recirculating air-handling systems that includes a simple prescriptive requirement for calculating air-handling system ventilation efficiency (which in turn is used to determine the air-handler minimum outdoor air rate) as well as minimum set points for VAV zones. The new procedure is as follows:

6.2.5.3 Simplified Procedure

6.2.5.3.1 System Ventilation Efficiency. System ventilation efficiency (E_v) shall be determined in accordance with Equation 6.2.5.3.1a or 6.2.5.3.1b.

$$E_v = 0.88 \times D + 0.22 \quad \text{for } D < 0.60 \quad (6.2.5.3.1a)$$

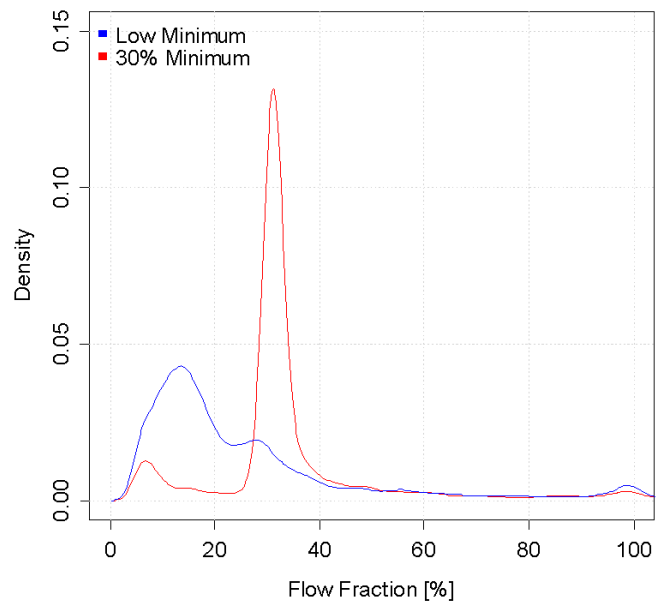
$$E_v = 0.75 \quad \text{for } D \geq 0.60 \quad (6.2.5.3.1b)$$

6.2.5.3.2 Zone Minimum Primary Airflow. For each zone, the minimum primary airflow (V_{pz-min}) shall be determined in accordance with Equation 6.2.5.3.2.

$$V_{pz-min} = V_{oz} \times 1.5 \quad (6.2.5.3.2)$$

This Simplified Procedure may be used in lieu of the more comprehensive procedure using the MSE from Standard 62.1, Appendix A. To determine VAV zone minimum set points using the MSE is very complicated, involving many assumptions about occupancy and airflow rates and requiring multiple iterations to find the critical zone, adjust the system outdoor air rate and/or zone minimum, and repeat until the perceived lowest energy solution is reached. This complexity and uncertainty is now eliminated with this addendum.

Shortly after the Standard 62.1 addendum was published, SSPC 90.1 passed Addendum au to Standard 90.1-2016. This addendum mandates the use of the new Simplified Procedure minimum in Equation 6.2.5.3.2 above in place of the previous provision in Standard 90.1-2016 that allowed VAV box minimum set points to be 20% of the design supply



air rate. Outdoor air rates are generally much lower than 20% of the maximum rate, but designers felt they needed a higher percentage to meet the requirements of Standard 62.1 for multiple zone systems. With this addendum, designers no longer need to calculate what minimum rates are required using the MSE. Moreover, using percentages to determine minimums is problematic because VAV boxes are almost always oversized due to conservative load assumptions for occupants, lights, plug loads, etc. It is not unusual for boxes to be sized three or more times larger than they need to be, as was found to be the case in ASHRAE RP-1515, Thermal and Air Quality Acceptability in Buildings that Reduce Energy by Reducing Minimum Airflow from Overhead Diffusers. The figure above from RP-1515 shows measured frequency of airflow rates in seven California office buildings using 30% minimums (based on earlier versions of Standard 90.1 and Title 24 Energy Standards) compared to the current dual maximum logic required by both Standard 90.1 and Title 24 for systems with DDC using the minimum set to the Title 24 minimum ventilation rate. (The Title 24 minimum ventilation rate is similar to the rates that result from Addendum f to Standard 62.1 described earlier.) The figure shows that, even if the minimums were set to 20% instead of 30%, excess air would have been supplied due to the oversized cooling maximum set point, wasting fan energy, heating energy, and cooling energy.

This addendum to Guideline 36 implements these two Standard 62.1 and 90.1 addenda to allow minimum rates to be calculated automatically from zone occupant-based and area-based ventilation rates. Energy savings are enhanced by further adjusting rates dynamically based on zone ventilation effectiveness (E_z) as a function of zone supply air temperature and occupancy status using occupancy sensors and CO₂ sensors. Similarly, minimums can be automatically calculated using Title 24 ventilation rates for California projects.

The proposed procedure still allows users to override the automatic calculation and enter a specific airflow rate for special circumstances:

- For fan-powered VAV boxes, the primary airflow minimum can often be set below the minimum ventilation rate by using transfer air to provide some or all of the ventilation requirement indirectly, as allowed by Standard 62.1, Appendix A.
- Minimum rates are sometimes determined by exhaust makeup air requirement rather than ventilation rates.

This addendum also contains clarifications and bug fixes regarding V_{bz-A} and V_{bz-P} , which are the active area- and people-based ventilation rates adjusted for occupancy sensors and window switches. They were not properly referenced in Section 5.16.3.1.

Note: In this addendum, changes to the current standard are indicated in the text by underlining (for additions) and ~~striking through~~ (for deletions) unless the instructions specifically mention some other means of indicating the changes.

Addendum e to Guideline 36-2018

Revise Section 3.1.2.1 as shown.

3.1.2.1 VAV Cooling-Only Terminal Unit

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$).
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

~~For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{asp}$. For calculation of $MinO_{asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV, for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).~~

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 3.1.2.2 as shown.

3.1.2.2 VAV Reheat Terminal Unit

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$).
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

~~For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the~~

~~highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{asp}$. For calculation of $MinO_{asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV, for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).~~

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 3.1.2.3 as shown.

3.1.2.3 Parallel Fan-Powered Terminal Unit, Constant-Volume Fan

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$).
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

~~For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{asp}$. For calculation of $MinO_{asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV, for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).~~

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 3.1.2.4 as shown.

3.1.2.4 Parallel Fan-Powered Terminal Unit, Variable-Volume Fan

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$).
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{Asp}$. For calculation of $MinO_{Asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV, for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 3.1.2.5 as shown.

3.1.2.5 Series Fan-Powered Terminal Unit, Constant-Volume Fan

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$)
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{Asp}$. For calculation of $MinO_{Asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

[. . .]

3.1.2.6 Series Fan-Powered Terminal Unit, Variable-

Revise Section 3.1.2.6 as shown.

Volume Fan

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$)
- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO

is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{Asp}$. For calculation of $MinO_{Asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$, except for zones that have CO_2 DCV, for which V_{min} should be equal to $V_{area-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 3.1.2.7 as shown.

3.1.2.7 Dual-Duct VAV Terminal Unit

- a. Zone maximum cooling airflow setpoint ($V_{cool-max}$)

For ASHRAE Standard 62.1 ventilation, select V_{min} to prevent creating critical zones. Critical zones are zones with the highest zone primary outdoor air fraction Z_{pz} , which results in lower system ventilation efficiency E_v and higher effective minimum outdoor air set point $MinO_{Asp}$. For calculation of $MinO_{Asp}$ for Standard 62.1 ventilation, see Section 5.16.3.1. This will lead to V_{min} being higher than code minimum ventilation for all zones that require outdoor air. For California Title 24 ventilation, V_{min} should be selected as the larger of $V_{area-min}$ and $V_{occ-min}$. When selecting V_{min} , do not consider the limitations of the VAV box controller to measure and control airflow; that is addressed by the control sequences themselves (see Section 5.1.16).

- b. Zone minimum airflow setpoint (V_{min}). This is an optional entry. If no value is scheduled, or a value of AUTO is scheduled, V_{min} will be calculated automatically and dynamically to meet ventilation requirements.

In most cases, V_{min} should be allowed to be automatically calculated. This ensures compliance with Standard 62.1 and Standard 90.1 prescriptive requirements and with California's Title 24 Energy Standards requirements, and it results in the lowest energy costs.

Revise Section 5.2.1.3 as shown.

5.2.1.3 For compliance with the Ventilation Rate Procedure of ASHRAE Standard 62.1-2016, outdoor air and zone minimum setpoints shall be calculated as follows:

- a. See Section 3.1.1.2 for zone ventilation setpoints.

- b. Determine zone air distribution effectiveness E_z .
1. If the discharge air temperature at the terminal unit is less than or equal to zone space temperature, E_z shall be equal to E_zC (default to 1.0 if no value is scheduled).
 2. If the discharge air temperature at the terminal unit is greater than zone space temperature,
 3. E_z shall be equal to E_zH (default to 0.8 if no value is scheduled).
- c. V_{bz-P^*} is the population component of the required breathing zone outdoor airflow. The normal value of V_{bz-P^*} shall be V_{bz-P} . V_{bz-A^*} is the area component of the required breathing zone outdoor airflow. The normal value of V_{bz-A^*} shall be V_{bz-A} .
- d. V_{min}
1. shall be equal to V_{oz} as calculated in Section 5.2.1.3(f) if V_{min} in Section 3.1.2 is AUTO and the associated air handler has been supplying 100% outdoor air (outdoor air damper fully open; return air damper fully closed) for 10 minutes;
 2. else shall be equal to $1.5 * V_{oz}$ as calculated in Section 5.2.1.3(f) if V_{min} in Section 3.1.2 is AUTO and the associated air handler is not supplying 100% outdoor air;
 3. else shall be equal V_{min} as entered in Section 3.1.2.
- e. The occupied minimum airflow V_{min}^* shall be equal to V_{min} except as noted in Section 5.2.1.3(e-f).
- f. The required zone outdoor airflow V_{oz} shall be calculated as $V_{oz} = (V_{bz-A^*} + V_{bz-P^*})/E_z$, where the normal values of V_{bz-A^*} and V_{bz-P^*} are modified if any of the following conditions are met, in order from higher to lower priority:
1. If the zone is in any mode other than occupied mode, and for zones that have window switches and the window is open: $V_{bz-P^*} = 0$, $V_{bz-A^*} = 0$, and $V_{min}^* = 0$.
 2. If the zone has an occupancy sensor, is unpopulated, and occupied-standby mode is permitted: $V_{bz-P^*} = 0$, $V_{bz-A^*} = 0$, and $V_{min}^* = 0$.
 3. Else, if the zone has an occupancy sensor, is unpopulated, but occupied-standby mode is not permitted: $V_{bz-P^*} = 0$ and $V_{min}^* = V_{min}$.

Revise Section 5.2.1.4 as shown. Renumber the remaining sections accordingly.

5.2.1.4 For compliance with California Title 24, outdoor air set points shall be calculated as follows:

- a. See Section 3.1.1.2.2 for zone ventilation set points.
- b. Determine the zone minimum outdoor air set points Zone-Abs-OA-min and Zone-Des-OA-min.

Zone-Abs-OA-min is used in terminal-unit sequences and air-handler sequences. Zone-Des-OA-min is used in air-handler sequences only.

1. Zone-Abs-OA-min shall be reset based on the following conditions in order from highest to lowest priority:
 - i. Zero if the zone has a window switch and the window is open.
 - ii. Twenty-five percent (25%) of $V_{area-min}$ if the zone has an occupancy sensor and is unpopulated.

The term "populated" is used instead of "occupied" to mean that a zone occupancy sensor senses the presence of people, because the term "occupied" is used elsewhere to mean "scheduled to be occupied."

- iii. $V_{area-min}$ if the zone has a CO_2 sensor.
 - iv. Zone-Des-OA-min otherwise.
2. Zone-Des-OA-min is equal to the following:
 - i. Zero if the zone has a window switch and the window is open.
 - ii. Twenty-five percent (25%) of $V_{area-min}$ if the zone has an occupancy sensor and is unpopulated.
 - iii. The larger of $V_{area-min}$ and $V_{occ-min}$ otherwise.

c. V_{min}

1. shall be equal to Zone-Abs-OA-min if V_{min} in Section 3.1.2 is AUTO;
2. else shall be equal to V_{min} as entered in Section 3.1.2.

Revise Section 5.16.3.1 as shown.

- c. Outdoor air absolute minimum and design minimum set-points are recalculated continuously based on the ~~mode~~ adjusted ventilation rates V_{bz-A^*} and V_{bz-P^*} of the zones being served, determined in accordance with Section 5.2.1.3.

Some diversity factor is included in V_{ou} calculated below because the ventilation requirements have been zeroed out for unoccupied zones and those with open window switches. But there is additional diversity in areas with occupancy sensors because only one person in the room will trigger the sensor. There is also diversity in other areas without occupancy sensors. Hence operating V_{ou} is limited to design V_{ou} and the diversity value of D in the calculation of $DesV_{ou}$ is not required.

1. Calculate the uncorrected outdoor air rate V_{ou} for all zones in all Zone Groups that are in Occupied Mode, but note that V_{ou} shall be no larger than the design uncorrected outdoor air rate, $DesV_{ou}$.

~~$$V_{ou} = \text{MIN}[DesV_{ou} | (\sum V_{bz-A} + \sum V_{bz-P})]$$~~

$$V_{ou} = \text{MIN}[DesV_{ou} | (\sum V_{bz-A^*} + \sum V_{bz-P^*})]$$

[...]

POLICY STATEMENT DEFINING ASHRAE'S CONCERN FOR THE ENVIRONMENTAL IMPACT OF ITS ACTIVITIES

ASHRAE is concerned with the impact of its members' activities on both the indoor and outdoor environment. ASHRAE's members will strive to minimize any possible deleterious effect on the indoor and outdoor environment of the systems and components in their responsibility while maximizing the beneficial effects these systems provide, consistent with accepted Standards and the practical state of the art.

ASHRAE's short-range goal is to ensure that the systems and components within its scope do not impact the indoor and outdoor environment to a greater extent than specified by the Standards and Guidelines as established by itself and other responsible bodies.

As an ongoing goal, ASHRAE will, through its Standards Committee and extensive Technical Committee structure, continue to generate up-to-date Standards and Guidelines where appropriate and adopt, recommend, and promote those new and revised Standards developed by other responsible organizations.

Through its *Handbook*, appropriate chapters will contain up-to-date Standards and design considerations as the material is systematically revised.

ASHRAE will take the lead with respect to dissemination of environmental information of its primary interest and will seek out and disseminate information from other responsible organizations that is pertinent, as guides to updating Standards and Guidelines.

The effects of the design and selection of equipment and systems will be considered within the scope of the system's intended use and expected misuse. The disposal of hazardous materials, if any, will also be considered.

ASHRAE's primary concern for environmental impact will be at the site where equipment within ASHRAE's scope operates. However, energy source selection and the possible environmental impact due to the energy source and energy transportation will be considered where possible. Recommendations concerning energy source selection should be made by its members.

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